

CALIBRATION TEST
OF ELECTRIC WATER METER

BY

W. H. STEWARD, JR.

ARMOUR INSTITUTE OF TECHNOLOGY

1915

628.17

SI 3



**Illinois Institute
of Technology
UNIVERSITY LIBRARIES**

AT 394
Steward, W. H.,
Calibration test of electric
water meter

For Use In Library Only



CALIBRATION TEST OF ELECTRIC WATER METER

A THESIS

PRESENTED BY

WM. H. STEWARD, Jr.

TO THE

PRESIDENT AND FACULTY

OF

ARMOUR INSTITUTE OF TECHNOLOGY

FOR THE DEGREE OF

BACHELOR OF SCIENCE IN MECHANICAL ENGINEERING

HAVING COMPLETED THE PRESCRIBED COURSE OF STUDY IN

MECHANICAL ENGINEERING

W. T. Seibhardt
L. E. ...
J. M. Raymond

6-15
7-12

TABLE OF CONTENTS .

	Page
Object of test	1
Description of apparatus	
(a) Multi-opening Averaging Pitot tube	1
(b) Meter or manometer	2
(c) Electrical apparatus	5
General scheme	5
Source of Power	6
Test of Electrical Contacts	7
Preliminary tests and difficulties	8
Method of procedure in final test	10
Tables and curves	12
Theory and formulae	14
Discussion of results	16
Conclusions	18
Bibliography	19

1. The first part of the document discusses the importance of maintaining accurate records of all transactions and activities. It emphasizes that proper record-keeping is essential for ensuring transparency and accountability in financial operations.

2. The second part of the document outlines the various methods and techniques used to collect and analyze data. It highlights the need for consistent and reliable data collection processes to support informed decision-making.

3. The third part of the document focuses on the analysis and interpretation of the collected data. It discusses the various statistical and analytical tools used to identify trends, patterns, and anomalies in the data.

4. The fourth part of the document discusses the importance of communication and reporting in the context of data analysis. It emphasizes the need for clear and concise communication of findings to stakeholders and the importance of providing actionable insights.

5. The fifth part of the document discusses the challenges and limitations of data analysis and provides strategies to overcome them. It highlights the need for continuous learning and improvement in data analysis practices.

6. The sixth part of the document discusses the future of data analysis and the emerging trends in the field. It highlights the importance of staying up-to-date with the latest developments and technologies in data analysis.

7. The seventh part of the document discusses the ethical considerations and privacy concerns associated with data analysis. It emphasizes the need for responsible and ethical data handling practices.

8. The eighth part of the document discusses the role of data analysis in various industries and sectors. It highlights the wide range of applications and the impact of data analysis on business operations and decision-making.

9. The ninth part of the document discusses the importance of data literacy and the need for individuals to have the skills and knowledge to effectively use and interpret data.

10. The tenth part of the document discusses the importance of data governance and the need for organizations to have clear policies and procedures in place to ensure the proper use and management of data.

T A B L E O F C O N T E N T S (continued)

Data on Electrical contacts	Table	1
Data for Venturi Flow Curve	"	2
Data and results from run 1	"	3
Data and results from run 2	"	4
Data and results from run 3	"	5
Data and results from run 4	"	6
Data and results for run 5	"	7
Calibration curve for Electrical contacts, plate		1
Dynamic Head Curve for Pitot tube	"	2
Calibration Curve for Venturi meter	"	3
Ampere Flow Curve for run 1	"	4
Ampere Flow Curve for run 2	"	5
Ampere Flow Curve for run 3	"	6
Ampere Flow Curve for run 4	"	7
Ampere Flow Curve for run 5	"	8
Coefficient Flow Curve for run 1	"	9
Coefficient Flow Curve for run 2	"	10
Coefficient Flow Curve for run 3	"	11
Coefficient Flow Curve for run 4	"	12

T A B L E O F C O N T E N T S (continued)

Coefficient Flow Curve for run 5	plate	13
The multi-opening averaging pitot tube	"	14
Section through Meter	"	15-16
Scheme of Connections for test	"	17

*****:*

The object of this report is the investigation of a new type of water meter.

The meter consists of a combination of three different parts, namely, the part inserted into the pipe line from which the static and dynamic heads are obtained, the manometer in which the dynamic head is measured, and the electrical apparatus used in measuring the head produced. The meter was designed for use in measuring the flow of water in a three inch standard pipe.

The part of the meter which is inserted into the pipe consists of a multi-opening averaging pitot tube, which is constructed as shown in plate 14, with two tubes, AA, 3.06 inches in length, 1-4 inch outside diameter, having ten 1-16" holes drilled as shown. . These tubes extend from a solid cylindrical part, B, which is 5-8 of an inch in diameter and 2 3-4 inches long, through which are two holes bored lengthwise, corresponding to the bore of the tubes extending from it. The other end of this cylindrical piece is arranged with separate stop cocks, CC, for each tube

1. 1911

1912

2.

. and threaded for connection to pipe union. This part is fitted into a stuffing box, D, which is arranged so that it can be clamped securely to the inner part by means of a screw, E. The stuffing box is threaded to fit in connections for 3-4 inch pipe.

This is screwed into a 3-4 inch hole in the pipe and the tubes AA, are turned so that the holes in the dynamic side face upstream and those in the static side face downstream. The tubes reach to the other side of the pipe and the stuffing box is screwed into the pipe so that it just reaches to the inner side of the pipe. The clamping screw is then tightened to hold it in place.

The manometer, or part in which the dynamic head is measured, is shown in plates 15 and 16. The connection with the static and dynamic sides of the Pitot tube are made at A and B respectively. The valve C closes the connection between the two sides and causes the mercury column to rise in the body of the manometer. The pipe G, 1-4 inch standard,

connects the static pressure with the chamber E, which is filled with oil. The pipe D, 1-4 inch standard, connects the dynamic pressure with the chamber F, which is filled with mercury to the level O. The oil in E overflows into K, and comes in contact with the surface of the mercury. The difference in pressure between F and E causes the mercury to rise in K, thus acting as a manometer. The mercury level is adjusted so that it just touches the end of the longest of the 41 contacts, L, when the slightest head is obtained above the zero. These contacts are attached to resistances which vary a definite amount for each wire. The resistance is in the form of coils wound around spools which are held between the non-conducting discs, N and P. The wires vary in length uniformly through a range of 21 of an inch, giving a
64
different resistance for the very small difference in head of .0082 of an inch of mercury. A rubber ring, R, is arranged so that each wire passes through it. This holds the wires apart and prevents short circuiting

...

...

...

...

...

...

...

...

...

...

...

...

...

...

...

...

...

...

...

...

...

...

4.

between the wires themselves and also between the wires and the wall S. The contacts, resistances, etc., are connected together in one piece which is suspended, as shown in plate 15, by a screw, having fine threads, which passes through the plug T in the top of the meter. The meter is grounded. Under these conditions no current flows until the contacts, L, come in contact with the mercury in the chamber F. The amperes depend upon the resistance in the circuit, which depends upon the number of contacts touching the mercury. The latter depends upon the height of the mercury in chamber F, which depends upon the difference between combined static and dynamic head and the static head. This difference in head bears a direct relation to the velocity or flow of water in the pipe. The body of the meter is made of cast iron, of shape as shown. The cover is bolted on and is provided with a projecting ring which fits into a corresponding groove in the body itself. A gasket insures an air tight joint between cover and body of meter. A plug, U, is

provided for draining chamber E, and one also as V, to drain the chamber K. The plug W, is used to draw out enough mercury to adjust the level for the zero reading.

The electrical apparatus used in measuring the current consists of an A. C. Voltmeter (0-60), a wattmeter, transformer, a coil for varying the resistance of the current, and an A. C. Ammeter (0-2). Direct current cannot be used because of the plating effect of the mercury upon the contacts. The diagram of the electrical connections is shown in plate 17. When the relation between the amperes and flow are found, the flow in the pipe can be found by referring to the ammeter. The wattmeter is used for continuous readings.

The scheme of connections used in the final tests is shown in plate 17. The meter was attached to a three inch line containing a Venturi meter with a mercury manometer reading in inches of mercury. The pitot tube was placed in the side of the horizontal

run of pipe so that the flow was measured after passing the Venturi meter. A gate valve was provided in the vertical pipe as shown, and a needle valve leading to a Pelton wheel was used in regulating the flow through the pipe. The meter was grounded through the 3" pipe, alternating current was used from a rotary converter at 75 volts. This was reduced to 37.5 volts at the transformer. One line from the transformer was grounded and the ammeter, and resistance coil was connected in the other. The resistance coil was used in regulating the current through the meter. This line was attached to the meter

The water was obtained from a well in the hydraulic laboratory, and was discharged into this same well. The water was supplied to the line by a multi-stage Worthington centrifugal pump, direct connected to a 40 H. P. Kerr turbine situated in the engine room. The suction to this pump was carried overhead from the well and an air pocket was formed in this line. This line was primed by a line from

7.

the city water mains and an air cock was provided at its highest point for the escape of the air.

Alternating current was obtained from a rotary converter, in the Dynamo laboratory, receiving direct current from the mains at 110 volts and delivering 75 volts alternating current. This was transformed down to 37.5 volts at the meter.

The electrical contacts were tested in order to find out the ammeter reading for each electrical contact, and to determine the distance between the ends of each two contacts. This was done by connecting each contact separately, in series with the circuit and taking the ammeter reading for each. The short circuit ammeter reading was adjusted to .97 by changing the resistance and the test was made with the line voltage contact at 37.5. The ammeter reading for each contact was tabulated and the difference in head between the longest (No.1) contact and each successive one was tabulated. The results of this test are shown in table 1 and from this data the

curves in plates 1 and 2 were drawn, showing the ammeter reading for each contact, and for each corresponding variation in dynamic head.

Numerous preliminary tests were made upon the meter and several adjustments and changes were made before consistent results were obtained. The meter was at first attached to the 3" boiler feed line, which was supplied with a Venturi meter having an indicating and recording apparatus. The test was made with the other apparatus arranged as shown in plate 17. The velocity was low in this line and the flow variable. The variation in the velocity in the line, due to the pulsating effect of the reciprocating boiler feed pumps, caused the ammeter reading to be continually changing. This made it hard to get accurate ammeter readings. The meter was then changed to the position on the line as shown in plate 17. The flow in this line from a centrifugal pump, was steady and the velocity could be regulated as desired. The results of runs on this line showed

that something was wrong. It was found that the rubber ring, shown as R, in plate 15, had slipped down and some of the contacts had touched each other, thereby causing an error in the results obtained. This ring was placed as it should have been and more consistent results were obtained, but it was found that in some way mercury had gotten into the lower part of the chamber E and water had gotten in between the mercury and oil in K. The meter was cleaned and dried and the mercury was placed in chamber F to a height a little above the zero level. Oil was then poured into E and K until the meter was filled to the cover with oil. The top was bolted on, and when the mercury level was adjusted properly, the readings were consistent. The electrical contacts and coils were at first held in a stationary position between the cover of the meter and the upper edge of wall S. This was changed by attaching this part to a screw as described before and allowing space of about 1-8 of an inch for the contacts to be raised or lowered by

10.

turning the screw at the top. This permitted the contacts to be adjusted to suit the level of the mercury in the meter. A mercury manometer was connected to the meter at the points A and B after the plugs H and J, plate 15, had been removed. This allowed the head to be read directly, but as trouble resulted in removing all the air from the system, and also as the variations in head were so very small, this arrangement was not used. The final tests were made after all of these changes had been made.

In starting the test a number of things had to be considered. The suction to the pump was carefully primed and the centrifugal pump started. The water was allowed to flow for some time through the pipe before readings were taken because of the great quantity of air in the pipe at first. After the air had been driven out of the line, the air was let out of the pockets existing in the connections to the Venturi meter and the electric meter.

11.

By opening the valve C, plate 15, between the static and dynamic sides of the meter, a flow of water due to the difference in pressure drives out the air from these connections. After the air had been driven from the connections, the ammeter and other electrical connections were arranged as in plate 17. The rotary converter was started and the meter was ready for adjustment. The valve C was closed and the meter short circuited by allowing all of the contacts to touch the mercury and the ammeter reading was adjusted to .97 by varying the resistance R. The voltage was read across the transformer terminals to be 37.5. The mercury and contacts were then adjusted to zero. This was accomplished by closing the needle valve, thereby stopping the flow, and adjusting the level of the contacts to their highest position and then drawing out enough mercury from plug W to bring the ammeter reading to zero. The level of the contacts was then adjusted, by means of the screw at the top, so that the slightest rise in the mercury caused a

reading of the ammeter corresponding to that of the lowest contact. The meter was then ready for use. Runs were then made, starting with no flow and increasing to the highest and back again, varying the flow just enough to give every change of ammeter reading. The corresponding head on Venturi manometer was recorded. The voltage across the terminals of the transformer was read at intervals to note any variation in the voltage in the supply line. The results of these runs are shown in the tables 3, 4, 5, 6, 7.

From the data obtained in the different parts of the tests the results were tabulated and curves drawn. Table 1 gives the data obtained on the electrical contacts. These results were used in making the curves shown in plates 1 and 2. Plate 1 shows the ammeter reading for each contact, and plate 2 shows the corresponding difference in head for each ammeter reading. Table 2 contains the flow Q , in cubic feet per second, for each variation of one inch of mercury

13.

in the head h , as indicated by the Venturi meter.

The value of Q was calculated from the formula for this meter, which is $Q = .069 \sqrt{h}$. from this data

the flow curve for the Venturi meter was made as shown in plate 3. Tables 3, 4, 5, 6, and 7 were made from the data obtained in the final runs upon the meter. The ammeter and Venturi head readings were taken in the tests and tabulated.

The pitot dynamic head in inches of mercury was obtained from the dynamic head curve in plate 2. From this data the flow in cubic feet per second was found from each meter and the coefficient for each reading obtained by dividing the flow as found by the electric meter by that found from the Venturi meter. The average of these runs was found. Plates 4, 5, 6, 7 and 8 show the ammeter reading for each variation in flow for each run. The flow was taken as that indicated by the Venturi meter. Plates 9, 10, 11, 12 and 13 show the values for the coefficient for each variation in flow obtained from the Venturi



meter for each run. The mean value of the coefficient is shown in each case.

In obtaining the results shown in the tables from the original data, a few simple formulas were required.

These formulas are those found by experiment to suit the various uses. In obtaining the flow in cubic feet per second as indicated by the pitot tube, the theoretical formula was used. This formula has been found to meet the conditions found in experiment. The ammeter reading gives the head, by referring to plate 2, and the corresponding velocity is obtained from the formula $V = \sqrt{2gh}$ where V is the velocity in feet per second, g is the acceleration due to gravity and h is the average dynamic head in feet of water. The multi-opening averaging pitot tube was assumed to give the average head. This tube has been described above. The ten holes receiving the impact of the moving water are spaced across the pipe as follows:

The area of the pipe is divided into five equal areas and a circle is drawn in each at a position

such that each area is divided in half. The impact on each of these circles gives the average for each area and the head corresponding to the average velocity of flow across the area of the pipe is produced by averaging these impacts. The formula for finding the position of each of these holes is

$$R = r \sqrt{\frac{2a-1}{N}} \quad \text{where}$$

R = distance from the center of the pipe to each position of pitot opening,

a = the number of the position from center of pipe

N = the total number of positions taken across pipe

r = radius of pipe in inches.

In this case, $r = 1-2 \times 3.06'' = 1.53''$ and $N = 10$.

$$R = 1.53 \sqrt{\frac{1}{10}} = .48''$$

$$R = 1.53 \sqrt{\frac{3}{10}} = .83''$$

$$R = 1.53 \sqrt{\frac{5}{10}} = 1.07'' \text{ etc.}$$

Knowing the value of the head, h and calculating the value of the velocity V , in the pipe, the flow in cubic feet per second is found by the simple multipli-

cation of the velocity by the area of the pipe which

$$\text{makes } Q = \frac{\pi}{4} d^2 v = \frac{\pi}{4} d^2 \sqrt{2gh},$$

$$g = 32.2 \quad d = 3.06" \text{ for standard 3" pipe.}$$

The results obtained in the various parts of the test show that the meter is very sensitive. The curves in plates 1 and 2 show that the resistances attached to the contacts give a curve for amperes resembling a parabola. This curve is very smooth with the exception of one point between the readings .87 and .89 amperes. The calibration curve for the Venturi meter is drawn from a mathematical formula and represents a smooth parabola. The accuracy of the meter is known to be good. Plates 4, 5, 6, 7 and 8 showing the relation between the flow and ammeter reading show that there is a slight variation in the reading of the ammeter for an increasing flow and a decreasing flow. The decreasing flow in four out of the five cases shows a high ammeter reading for a given velocity in pipe. This is to be expected, owing to the capillary action of the walls from the upper

surface of the mercury. The average of these five curves is about the same, showing a reading of ammeter for each contact. The variations in these curves are also affected by slight changes in the line voltage. This was noted in table 6 which shows in curve on plate 7. The voltage increased near the end of the run, and the ammeter reading did not return to zero when the flow was stopped. This was unavoidable and it is fair to assume that the average of the results is correct because the test was run at the ordinary voltage carried and lower voltages no doubt existed during a part of these runs. The curves shown in plates 9, 10, 11, 12 and 13 show the relation between the coefficient and the flow. These curves show an increased coefficient for the decreasing flow. This set of curves is very irregular when compared to the ampere flow curves. The average coefficient for all of the runs is 1.25. Judging from the two sets of curves, the Venturi meter is less sensitive than the electric meter.



The following conclusions may be drawn with regard to the conditions influencing the accuracy and dependability of the electric meter as shown by this test:

1. The voltage must be constant in order to get constant ammeter readings for different rates of flow.

2. The mercury zero level must be accurately adjusted and its upper surface must be clean and free from dirt.

3. All air must be driven from the connections. The meter is therefore expected to give better service on lines which are constantly filled with water than on lines that may sometimes be drained and filled with air.

4. The meter may be depended upon to give consistent results.



BIBLIOGRAPHY .

Batten, Mr. Flow of gases, and pitot tubes.

Progressive Age, Dec. 1911. Tests made at Detroit Gas Company.

Bowie, A. J. Jr. Centrifugal fans. A. S. M. E.

1905 pages 217 to 231. Discussion of test using Pitot tube for measuring the flow of air. Fan losses described. Discussion by R. C. Carpenter, page 226, on accuracy of Pitot tube. 5 Curves.

Boyd, Jas. E. and Judd, Horace. Pitot tubes.

Experimental determinations of the forms and velocity of jets. Engineering News, March 31, 1904
Influence of shape of dynamic tube on the velocity reading shows that it does not matter what the shape is. 2 Curves. 2 Illustrations. 7 Tables.

Carpenter, R. C. Table of properties of air. Heating and Ventilation, page 381. Shows relative amounts of air and vapors at various temperatures at 29.921 inches of Mercury. Error in neglecting moisture is very small at ordinary temperatures.

- Durley, R. J. Actual discharge of air from round orifices. A. S. M. E. 1906. Pages 193-233. Table page 223, gives actual discharge of air from round orifices 5-16" diameter to 4 1-2" diameter, at heads from one to five inches of water. Conclusions page 221. 16 Curves. 2 Illustrations. 16 Tables.
- Eckart, W. R. Application of pitot tube to the testing of impulse water wheels. Institute Mechanical Engineers, Jan. 7, 1910.
- Method of finding mean velocity, mean pressure, and mean kinetic energy of a jet of water. Tables and diagrams of same. Form of pitot tube used. Head indicated by Bourdon gauge. 12 Curves. 3 Tables. 8 Illustrations.
- Freeman, John R. Hydraulics of fire stream. A. S. M. E. Vol. 21, Pages 414-415.
- Gregory, W. B. The Pitot tube. A. S. M. E. 1904. Pages 184-211. Description of principles and construction of pitot tubes. Good general

discussion of theory. 8. Illustrations 2. Curves.

Gregory, W. B. and Schoder, E. W. Some Pitot tube studies. A. S. M. E. 1908. Page 351.

Harza, L. F. Kinetic energy of flowing water.

Engineering News. March 1907. Says the maximum velocity in a jet is equal to the theoretical velocity calculated from head on nozzle.

Jager, O. E. and Westly, G. C. Velocity of gas in large conduits. A duct 12 by 15 feet in section was divided into 15 sections and the average of the readings taken in each section found. Alcohol was used in manometer. 3 Illustrations.

Kneeland, Frank H. Some experiences with the pitot tube on high and low air velocities. A. S. M. E. 1911. Pages 1137-1173. Description of types of pitot tubes used, and of automatic averaging manometer. Conclusions page 1155. (a) The Taylor tube very accurate and extremely sensitive up to air velocity of 3000 feet per minute. Is

clumsy, difficult to manipulate, fragile and altogether unsuited to high velocity measurement.

(b) The Gebhardt type is light, compact, very portable, convenient and easy to manipulate. Very accurate up to 6000 feet per minute but beyond this its accuracy is problematical.

(c) The U. S. No. 1 tube is less convenient and less portable than the Gebhardt. It is much simpler in construction and will stand more abuse. Does not require as much care in manipulation as the other two types. Accurate to 250000 feet per minute.

(d) The exact formula is more accurate, especially at high velocities, than the one usually used.

(e) Present data on friction of air in pipe are largely in error. Discussion by D. W. Taylor page 1168-1172. Attention called to importance of proper static openings. Impact upon static openings give too high reading. 37 Pages, 10

Illustrations. 8 Tables. 7 Curves.

Mc Coll, J. R. Tests of Vacuum cleaning systems.

A. S. M. E. 1913. Pages 705-730. Pitot tube method for determining air velocities was used.

Detail of pitot tube used. Same as used by C. H.

Treat. 3 Illustrations. 5 Curves. 1 Table.

Moss, Sanford A. Flow of air and other gases with reference to small pressure differences.

American Machinist Sept. 20, 27, 1906. Pitot tube used. Coefficients of discharge for orifices in thin plates. Formulæ for pitot tube and discharge of air derived.

Rankine, Flow of gases. Applied Mechanics page 579.

Rowse, W. C. Pitot tubes for gas measurement. A. S.

M. E. 1913. Pages 633-691. Discussion pages 692-703. Investigation of the reliability of the pitot tube as a means of measuring gases, and to ascertain which form of pitot tubes now in common use give incorrect results in the measurement of gases.



Pitot tube calibrated against a Thomas electric meter. Sketches of different pitot tubes used. Valuable information on various designs of static and impact openings for pitot tubes.

18 Illustrations. 16 Curves. 18 Tables.

Snow, W. B. Foundry blower practice A. S. M. E. 1908
Page 936. Discussion of the most satisfactory form of pitot tubes. 10 Tables 9 Curves. 6 Illustrations.

Taylor, D. W. Experiments with ventilating fans and pipes. Naval Architects and Marine Engineers now 1905. Design of pitot tube calculations.

Taylor, D. W. Some experiments with ventilating fans. Engineering News, Nov. 3d, 1904. Pages 388-390. Sketch of pitot tube used. Tests of ventilating fans on warships. Conclusions regarding friction in pipes.

Thomas, Carl C. The measurement of gases. Journal of the Franklin Institute, Nov. 1911. Pages 411-460. Discussion of formulae of Dr. Pole and Prof.

Unwin for the flow of gases through pipe lines. Volumetric and velocity types of gas meters described. The principles of the pitot tube Venturi meter, and the heat or electric meter given Comparative laboratory tests made upon the three types mentioned. Method of obtaining mean velocity head and point of mean velocity with pitot tube given. Comparative tests show that the pitot tube, Venturi meter, and electric meter give accurate and reliable results when properly used, under favorable conditions, and when observations are taken with a sufficient degree of refinement in methods. The electric meter gives continuous results, graphical and integrated, under severe commercial conditions, that compare favorably with the best that can be obtained with other forms.

Treat, Chas. H. Measurement of air in fan work.

A. S. M. E. 1912, Pages 1019-1054. Discussion of various types of pitot tubes and calibration of different impact and static orifices used in .

measurement of flow of air. Method of testing fans.

Friction in pipe. Standard weight of air used for convenience in comparing with other tests.

Average velocity of flow in pipe obtained by the concentric ring method (page 1036). Discussions page 1046 by W. H. Carrier on the effect of air leakage in static connection upon the fan efficiency. C. G. De Laval, page 1048, lays stress upon importance of considering humidity of air, 36 Pages. 14 Illustrations. 1 Table 2 Curves.

Weymouth, Thomas R. Problems in natural gas engineering. A. S. M. E. 1912. Pages 195-201. Derivation of new pipe line formula for the flow of natural gas. Test with pitot tube. 6 Pages. 2 Curves.

White, W. M. New method of reducing the readings of the pitot tube, and comparison with weir formula. American Machinist Vol. 29. Pages 175-179.

"Fans and Blowers." Power, May and June 1906.

Elementary article showing the application of
pitot tube.

"Pitometer." Proceedings Aug. Soc of W. Penn. Dec.
1906.

"Various forms of pitot tube with mathematical
discussion". Die Turbine. Jan. 20, 1910.

Table I

DATA ON ELECTRIC CONTACTS

Short Circuit Coverage = .97.

Difference in length between contacts No. 1 and 20 is .1131 in.

an inch = .328" 110 111/21 = .032"

Contact No.	Amperes.	Difference in length in Inches	Contact No.	Amperes	Difference in length in Inches
0	0	*	21	.75	.11
1	.10	.0082	22	.76	.11
2	.20	.0164	23	.79	.11
3	.25	.0255	24	.81	.11
4	.30	.0330	25	.84	.11
5	.35	.041	26	.86	.11
6	.50	.049	27	.87	.11
7	.60	.057	28	.88	.11
8	.75	.066	29	.89	.11
9	.87	.074	30	.90	.11
10	.95	.082	31	.91	.11
11	1.00	.090	32	.92	.11
12	1.05	.098	33	.93	.11
13	1.10	.107	34	.94	.11
14	1.20	.115	35	.95	.11
15	1.30	.123	36	.96	.11
16	1.40	.131	37	.97	.11
17	1.50	.140	38	.98	.11
18	1.60	.148	39	.99	.11
19	1.71	.156	40	1.00	.11
20	1.77	.161	41	1.07	.11

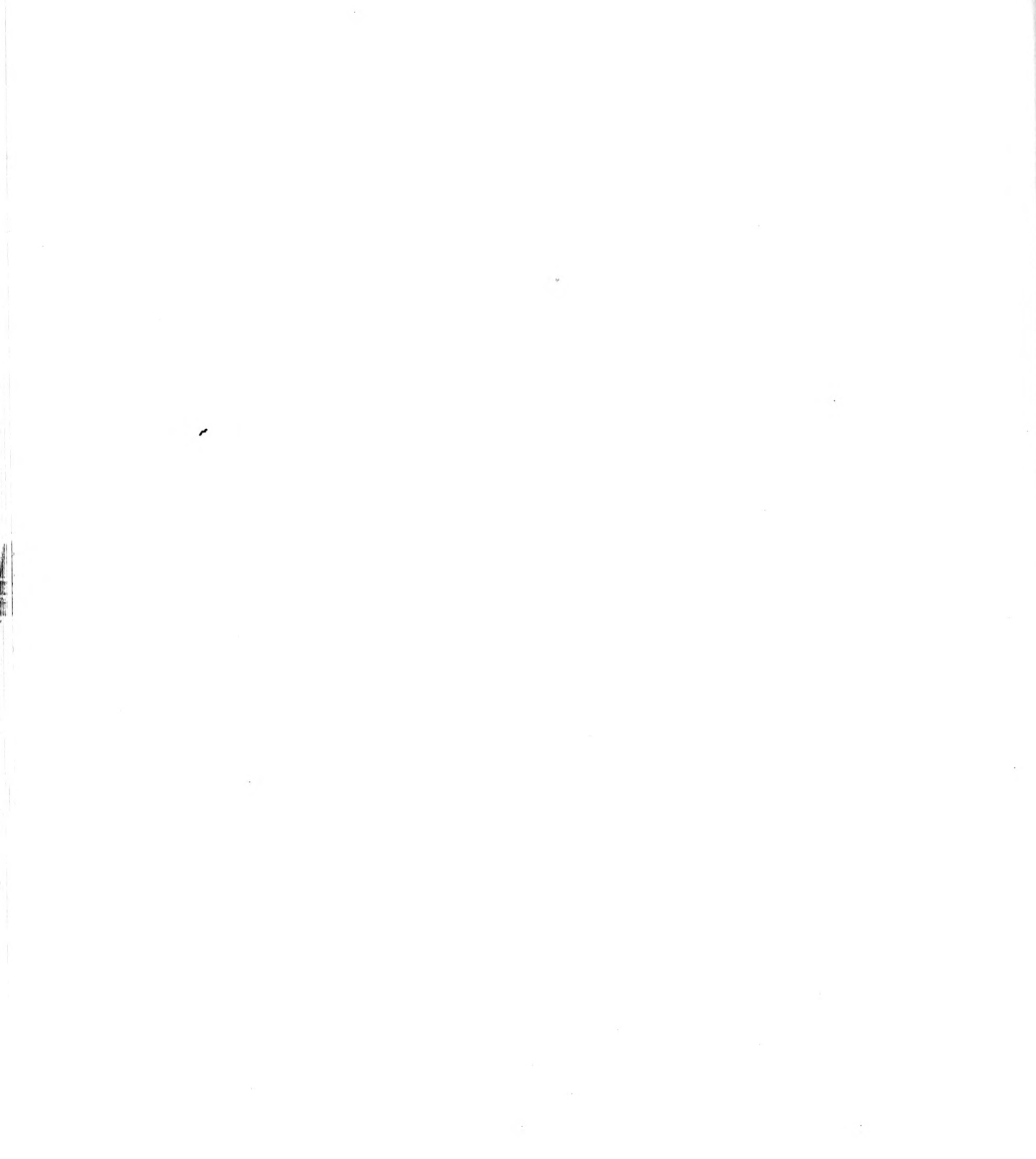
Table II.

Data for Venturi Flow Curve .

Q = flow in Cubic Feet per Second.

= $.069 \sqrt{h}$, where h = head in inches of Mercury.

H	Q	H	Q	H	Q	H	Q
0	0	5	.154	10	.218	15	.269
1	.069	6	.169	11	.229	16	.279
2	.0976	7	.1825	12	.239	17	
3	.1105	8	.195	13	.249	18	
4	.138	9	.207	14	.258	19	



Amperes	Venturi Head in Inches of Mercury	Pitot Dynamic Head in Inches of Mercury	Flow in Cubic Feet per Second From Venturi	Coefficient	Amperes	Venturi Head in Inches of Mercury	Pitot Dynamic Head in Inches of Mercury	Flow in Cubic Feet per Second From Venturi	Coefficient	Amperes	Venturi Head in Inches of Mercury	Pitot Dynamic Head in Inches of Mercury	Flow in Cubic Feet per Second From Venturi	Coefficient	Amperes	Venturi Head in Inches of Mercury	Pitot Dynamic Head in Inches of Mercury	Flow in Cubic Feet per Second From Venturi	Coefficient	Amperes	Venturi Head in Inches of Mercury	Pitot Dynamic Head in Inches of Mercury	Flow in Cubic Feet per Second From Venturi	Coefficient		
.1	0	0	0	0	.81	3.80	.0082	0	1.58	.84	3.80	0	1795	.107	.81	3.80	0	1795	.107	1.84	4.70	1.84	149	1735	1.165	
.2	.2	.0082	.035	1.58	.83	6.35	.0164	.035	.68	.83	6.35	.0164	173	.191	.83	6.35	.0164	173	.191	1.84	4.70	1.84	149	1735	1.165	
.3	.45	.0164	.0455	1.65	.81	6.00	.0225	.0455	1.77	.81	6.00	.0225	169	.18	.81	6.00	.0225	169	.18	1.84	4.70	1.84	149	1735	1.165	
.4	.5	.0225	.0475	1.77	.79	4.95	.0290	.065	1.48	.79	4.95	.0290	158	.158	.79	4.95	.0290	158	.158	1.84	4.70	1.84	149	1735	1.165	
.5	.55	.0290	.050	1.48	.73	4.70	.033	.074	1.51	.73	4.70	.033	149	.149	.73	4.70	.033	149	.149	1.84	4.70	1.84	149	1735	1.165	
.6	.60	.033	.0525	1.51	.70	4.40	.039	.079	1.375	.70	4.40	.039	145	.145	.70	4.40	.039	145	.145	1.84	4.70	1.84	149	1735	1.165	
.7	.60	.039	.0625	1.375	.68	4.20	.046	.086	1.255	.68	4.20	.046	141	.141	.68	4.20	.046	141	.141	1.84	4.70	1.84	149	1735	1.165	
.8	1.10	.046	.0745	1.255	.66	4.00	.0525	.0935	1.09	.66	4.00	.0525	138	.138	.66	4.00	.0525	138	.138	1.84	4.70	1.84	149	1735	1.165	
.9	1.20	.0525	.0775	1.09	.64	3.90	.0585	1.000	1.00	.64	3.90	.0585	136	.136	.64	3.90	.0585	136	.136	1.84	4.70	1.84	149	1735	1.165	
1.0	1.30	.060	.0880	1.00	.62	3.80	.066	1.067	1.21	.62	3.80	.066	134	.134	.62	3.80	.066	134	.134	1.84	4.70	1.84	149	1735	1.165	
1.1	1.55	.074	.095	1.21	.60	3.50	.074	1.185	1.25	.60	3.50	.074	129	.129	.60	3.50	.074	129	.129	1.84	4.70	1.84	149	1735	1.165	
1.2	1.80	.080	.104	1.25	.59	3.30	.080	1.237	1.10	.59	3.30	.080	128	.128	.59	3.30	.080	128	.128	1.84	4.70	1.84	149	1735	1.165	
1.3	2.05	.085	.105	1.10	.56	3.15	.085	1.270	1.21	.56	3.15	.085	126	.126	.56	3.15	.085	126	.126	1.84	4.70	1.84	149	1735	1.165	
1.4	2.30	.0925	.108	1.21	.54	3.00	.0925	1.297	1.22	.54	3.00	.0925	124	.124	.54	3.00	.0925	124	.124	1.84	4.70	1.84	149	1735	1.165	
1.5	2.45	.099	.110	1.22	.51	2.70	.099	1.329	1.22	.51	2.70	.099	122	.122	.51	2.70	.099	122	.122	1.84	4.70	1.84	149	1735	1.165	
1.6	2.60	.107	.110	1.22	.47	2.50	.107	1.455	1.230	.47	2.50	.107	120	.120	.47	2.50	.107	120	.120	1.84	4.70	1.84	149	1735	1.165	
1.7	2.80	.115	.120	1.230	.44	2.30	.115	1.480	1.235	.44	2.30	.115	118	.118	.44	2.30	.115	118	.118	1.84	4.70	1.84	149	1735	1.165	
1.8	3.00	.1175	.120	1.235	.41	2.15	.1175	1.490	1.15	.41	2.15	.1175	117	.117	.41	2.15	.1175	117	.117	1.84	4.70	1.84	149	1735	1.165	
1.9	3.25	.123	.125	1.15	.38	2.00	.123	1.530	1.15	.38	2.00	.123	115	.115	.38	2.00	.123	115	.115	1.84	4.70	1.84	149	1735	1.165	
2.0	4.00	.128	.1275	1.15	.37	1.80	.128	1.57	1.15	.37	1.80	.128	114	.114	.37	1.80	.128	114	.114	1.84	4.70	1.84	149	1735	1.165	
2.1	4.1	.137	.130	1.15	.35	1.65	.137	1.630	1.15	.35	1.65	.137	113	.113	.35	1.65	.137	113	.113	1.84	4.70	1.84	149	1735	1.165	
2.2	4.20	.150	.130	1.15	.35	1.45	.150	1.680	1.10	.35	1.45	.150	112	.112	.35	1.45	.150	112	.112	1.84	4.70	1.84	149	1735	1.165	
2.3	4.30	.156	.135	1.10	.33	1.30	.156	1.720	1.10	.33	1.30	.156	111	.111	.33	1.30	.156	111	.111	1.84	4.70	1.84	149	1735	1.165	
2.4	4.40	.168	.145	1.10	.33	1.15	.168	1.785	1.205	.33	1.15	.168	110	.110	.33	1.15	.168	110	.110	1.84	4.70	1.84	149	1735	1.165	
2.5	4.50	.180	.145	1.235	.32	1.05	.180	1.85	1.235	.32	1.05	.180	109	.109	.32	1.05	.180	109	.109	1.84	4.70	1.84	149	1735	1.165	
2.6	4.60	.194	.160	1.15	.29	0.95	.194	1.85	1.267	.29	0.95	.194	108	.108	.29	0.95	.194	108	.108	1.84	4.70	1.84	149	1735	1.165	
2.7	4.90	.199	.160	1.267	.28	0.90	.199	1.92	1.262	.28	0.90	.199	107	.107	.28	0.90	.199	107	.107	1.84	4.70	1.84	149	1735	1.165	
2.8	5.00	.217	.160	1.15	.28	0.85	.217	1.945	1.15	.28	0.85	.217	106	.106	.28	0.85	.217	106	.106	1.84	4.70	1.84	149	1735	1.165	
2.9	7.15	.217	.184	1.15	.27	0.80	.217	2.03	1.105	.27	0.80	.217	105	.105	.27	0.80	.217	105	.105	1.84	4.70	1.84	149	1735	1.165	
3.0	7.55	.220	.1895	1.262	.26	0.75	.220	2.045	1.08	.26	0.75	.220	104	.104	.26	0.75	.220	104	.104	1.84	4.70	1.84	149	1735	1.165	
3.1	7.10	.213	.183	1.15	.25	0.75	.213	2.015	1.10	.25	0.75	.213	103	.103	.25	0.75	.213	103	.103	1.84	4.70	1.84	149	1735	1.165	
3.2																										
3.3																										

Table IV.

Amperes	Venturi Head In Inches of Mercury	Pitot Dynamic Head in Inches of Mercury	Flow In Cubic Feet Per Second		Coefficient	Amperes	Venturi Head In Inches of Mercury	Pitot Head in Inches of Mercury	Flow In Cubic Feet Per Second		Coefficient
			From Venturi	From Pitot					From Venturi	From Pitot	
.0	0	.0	0	0		.83	6.35	.199	.173	.1942	1.128
.1	.40	.0082	.0425	.0395	.932	.81	5.90	.194	.167	.190	1.15
.2	.55	.0164	.050	.0557	1.11	.79	5.00	.187	.154	.188	1.02
.23	.65	.0225	.0645	.065	1.01	.72	4.60	.160	.1475	.1745	1.182
.27	1.00	.029	.069	.074	1.072	.69	4.35	.150	.143	.168	1.172
.29	1.10	.032	.0745	.078	1.05	.68	4.20	.1465	.141	.1635	1.18
.32	1.30	.0362	.080	.083	1.04	.66	4.00	.1375	.138	.162	1.105
.35	1.45	.041	.0855	.088	1.03	.64	3.90	.128	.136	.157	1.153
.39	1.85	.051	.0955	.0983	1.03	.62	3.70	.123	.1325	.153	1.153
.42	2.10	.057	.100	.104	1.04	.60	3.45	.1175	.128	.149	1.16
.44	2.20	.0645	.1025	.111	1.08	.58	3.45	.1105	.128	.1445	1.17
.47	2.35	.074	.106	.1185	1.12	.55	3.00	.099	.1195	.137	1.147
.49	2.60	.080	.112	.1237	1.10	.52	2.70	.0885	.114	.1307	1.14
.51	2.80	.085	.113	.127	1.096	.50	2.55	.082	.111	.1247	1.12
.52	2.80	.0925	.110	.1327	1.09	.48	2.50	.0795	.110	.122	1.11
.53	3.10	.103	.1275	.140	1.10	.45	2.20	.060	.1025	.114	1.11
.58	3.20	.111	.129	.1455	1.125	.42	1.95	.057	.0975	.104	1.065
.60	3.80	.1175	.134	.149	1.112	.40	1.75	.0525	.0925	.100	1.09
.62	4.00	.123	.138	.153	1.11	.36	1.50	.043	.0865	.0903	1.145
.63	4.10	.1375	.139	.162	1.168	.33	1.10	.0375	.0745	.0845	1.135
.69	4.40	.150	.1445	.163	1.16	.30	1.00	.033	.069	.079	1.145
.71	4.55	.156	.147	.172	1.17	.25	.80	.025	.0325	.060	1.105
.74	4.60	.168	.1475	.1785	1.21	.20	.60	.0164	.0525	.0557	1.06
.77	4.95	.1775	.153	.184	1.20	.15	.40	.0125	.0425	.0487	1.145
.79	5.10	.187	.155	.188	1.212	.10	.20	.0082	.025	.0365	1.58
.81	5.30	.194	.1585	.192	1.21	0	0	0	0	0	
.83	6.40	.199	.174	.194	1.12						
.85	6.70	.207	.178	.198	1.112						
.89	8.70	.258	.203	.2215	1.094						
.85	6.70	.207	.178	.198	1.112						
							Ave	Coef	=	1.123	

Table V.

Amperes	Venturi Head	Pitot Dynamic	Flow In Cubic	Coef- ficient	Amperes	Venturi Head	Pitot Dynamic	Flow In Cubic	Pitot Dynamic	Flow In Cubic	Coef- ficient
32.	38.	3880.	011.	VE51.	31.1	39.	381.	01.2	381.	381.	381.
33.	39.	390.	3911.	VE1.	271.1	39.	3911.	01.8	3711.	411.	411.
34.	30.	3011.	181.	VE44.	291.1	32.	3011.	02.8	301.	301.	301.
35.	32.	3111.	281.	VE41.	91.1	32.	281.	28.1	380.	3290.	3290.
36.	32.	321.	281.	VE1.	88.1	34.	281.	03.1	2870.	980.	881.
37.	34.	331.	3811.	VE11.	233.1	34.	3811.	04.1	960.	2380.	411.
38.	36.	341.	491.	VE1.	38.1	34.	491.	01.1	2520.	2470.	001.
39.	38.	351.	431.	VE1.	28.1	35.	431.	02.	140.	2220.	880.
40.	40.	361.	321.	VE11.	268.1	35.	321.	02.	330.	3320.	870.
41.	42.	371.	381.	VE1.	98.1	35.	381.	04.	350.	2240.	880.
42.	44.	381.	441.	VE1.	88.1	35.	441.	04.	4910.	2840.	7220.
43.	46.	391.	2441.	VE1.	212.1	35.	2441.	08.	3800.	230.	2920.
44.	48.	401.	3441.	VE1.	252.1	35.	3441.	08.	0	0	0
45.	50.	411.	491.	VE1.	42.1	35.	491.	08.	0	0	0
46.	52.	421.	5741.	VE1.	42.1	35.	5741.	08.	0	0	0
47.	54.	431.	421.	VE1.	38.1	35.	421.	08.	0	0	0
48.	56.	441.	021.	VE1.	28.1	35.	021.	08.	0	0	0
49.	58.	451.	081.	VE1.	28.1	35.	081.	08.	0	0	0
50.	60.	461.	271.	VE1.	28.1	35.	271.	08.	0	0	0
51.	62.	471.	321.	VE1.	28.1	35.	321.	08.	0	0	0
52.	64.	481.	381.	VE1.	28.1	35.	381.	08.	0	0	0
53.	66.	491.	441.	VE1.	28.1	35.	441.	08.	0	0	0
54.	68.	501.	501.	VE1.	28.1	35.	501.	08.	0	0	0
55.	70.	511.	561.	VE1.	28.1	35.	561.	08.	0	0	0
56.	72.	521.	621.	VE1.	28.1	35.	621.	08.	0	0	0
57.	74.	531.	681.	VE1.	28.1	35.	681.	08.	0	0	0
58.	76.	541.	741.	VE1.	28.1	35.	741.	08.	0	0	0
59.	78.	551.	801.	VE1.	28.1	35.	801.	08.	0	0	0
60.	80.	561.	861.	VE1.	28.1	35.	861.	08.	0	0	0
61.	82.	571.	921.	VE1.	28.1	35.	921.	08.	0	0	0
62.	84.	581.	981.	VE1.	28.1	35.	981.	08.	0	0	0
63.	86.	591.	1041.	VE1.	28.1	35.	1041.	08.	0	0	0
64.	88.	601.	1101.	VE1.	28.1	35.	1101.	08.	0	0	0
65.	90.	611.	1161.	VE1.	28.1	35.	1161.	08.	0	0	0
66.	92.	621.	1221.	VE1.	28.1	35.	1221.	08.	0	0	0
67.	94.	631.	1281.	VE1.	28.1	35.	1281.	08.	0	0	0
68.	96.	641.	1341.	VE1.	28.1	35.	1341.	08.	0	0	0
69.	98.	651.	1401.	VE1.	28.1	35.	1401.	08.	0	0	0
70.	100.	661.	1461.	VE1.	28.1	35.	1461.	08.	0	0	0

Table V.

Amperes	Venturi Head in Inches of Mercury	Pitot Dynamic Head in Inches of Mercury	Flow In Cubic Feet Per Second		Coefficient	Amperes	Venturi Head in Inches of Mercury	Pitot Dynamic Head in Inches of Mercury	Flow In Cubic Feet Per Second		Coefficient
			From Venturi	From Pitot					From Venturi	From Pitot	
0	0	0	0	0		.92	8.10	.270	.196	.2265	1.16
.1	.2	.0082	.025	.0395	1.58	.89	6.65	.258	.1775	.2215	1.35
.2	.4	.0164	.0425	.0557	1.31	.88	5.25	.246	.1575	.216	1.37
.25	.7	.025	.0575	.069	1.20	.86	4.90	.213	.152	.2015	1.35
.30	.85	.033	.0615	.079	1.22	.85	4.70	.207	.149	.198	1.33
.33	1.1	.0375	.0745	.0845	1.135	.82	4.40	.197	.1415	.194	1.34
.36	1.3	.043	.080	.0903	1.13	.80	4.30	.191	.1425	.190	1.33
.40	1.6	.0525	.089	.100	1.125	.78	4.10	.180	.139	.185	1.33
.42	1.75	.057	.0925	.104	1.125	.76	4.00	.175	.138	.182	1.31
.45	1.95	.069	.0975	.114	1.165	.74	4.00	.168	.138	.1765	1.29
.48	2.10	.0785	.100	.122	1.22	.72	3.80	.160	.134	.1745	1.33
.50	2.30	.082	.105	.1247	1.07	.67	3.40	.142	.1275	.164	1.29
.52	2.50	.0885	.110	.1297	1.18	.62	3.10	.123	.122	.153	1.25
.55	2.85	.099	.1165	.137	1.175	.60	2.70	.1175	.114	.149	1.33
.58	3.05	.1105	.121	.1445	1.195	.56	2.30	.103	.105	.140	1.33
.60	3.30	.1175	.125	.149	1.19	.50	1.85	.082	.0955	.112	1.37
.62	3.30	.123	.125	.153	1.22	.48	1.60	.0785	.099	.122	1.36
.65	3.45	.131	.126	.158	1.235	.45	1.40	.069	.0835	.114	1.34
.67	3.50	.142	.131	.164	1.25	.40	1.10	.0525	.0745	.100	1.32
.69	3.80	.150	.134	.168	1.25	.35	.90	.041	.0665	.088	1.50
.72	4.00	.160	.138	.1745	1.265	.30	.60	.033	.0525	.079	1.51
.74	4.00	.168	.138	.1785	1.29	.25	.45	.025	.0455	.068	1.31
.76	4.20	.175	.141	.182	1.29	.20	.40	.0164	.0425	.0557	1.58
.78	4.40	.180	.1445	.185	1.28	.10	.20	.0082	.025	.0395	
.80	4.40	.191	.1445	.190	1.315	0	0	0	0	0	
.82	4.50	.197	.146	.194	1.325						
.84	4.60	.205	.1475	.197	1.34		Ave	Coef	-	1.220	
.85	5.00	.207	.154	.198	1.285						
.88	5.40	.246	.160	.216	1.35						
.89	7.00	.258	.1825	.2215	1.215						

Table VI.

Amperes	Venturi Head in	Pitot Dynamic Head	Flow In Cubic Feet Per	Coef-ficient	Amperes	Ven-turi Head in	Pitot Dynamic Head	Flow In Cubic Feet Per	Coef-ficient	Amperes	Ven-turi Head in	Pitot Dynamic Head	Flow In Cubic Feet Per	Coef-ficient
32	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8
33	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8
34	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8
35	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8
36	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8
37	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8
38	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8
39	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8
40	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8
41	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8
42	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8
43	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8
44	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8
45	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8
46	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8
47	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8
48	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8
49	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8
50	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8	34	0.8	2011.	2111.	0.8

Flow in cubic feet per

pitot dynamic head

venturi head

amperes

coefficient

amperes

venturi head

pitot dynamic head

flow in cubic feet per

coefficient

amperes

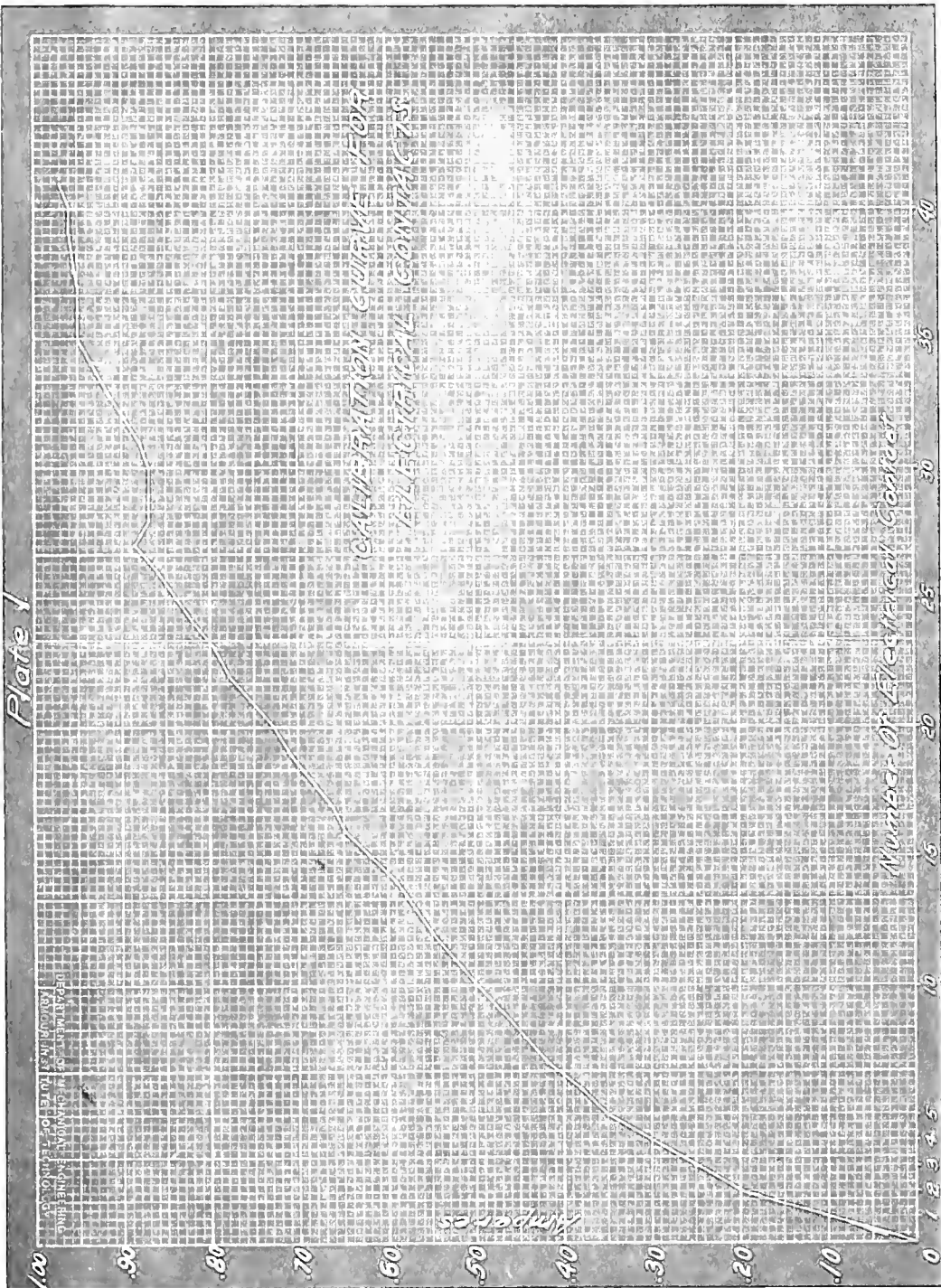
Table VI.

Amperes	Venturi Head in Inches of Mercury	Pitot Dynamic Head in Inches of Mercury	Flow In Cubic Feet Per Second		Coefficient	Amperes	Venturi Head in Inches of Mercury	Pitot Dynamic Head in Inches of Mercury	Flow In Cubic Feet Per Second		Coefficient
			From Venturi	From Pitot					From Venturi	From Pitot	
0	0	0	0	0		.84	4.40	.205	.1445	.197	1.301
.1	.20	.0082	.025	.0395	1.58	.82	4.30	.197	.1425	.194	1.330
.2	.35	.0164	.038	.0557	1.465	.79	4.10	.187	.139	.188	1.752
.25	.55	.025	.050	.069	1.33	.74	3.95	.168	.137	.1875	1.361
.30	.80	.033	.0625	.079	1.232	.72	3.65	.160	.132	.1745	1.325
.33	1.000	.0375	.069	.0845	1.22	.69	3.30	.150	.125	.168	1.34
.36	1.10	.043	.0745	.0903	1.21	.65	3.05	.131	.123	.158	1.305
.40	1.25	.0525	.0785	.100	1.272	.62	2.80	.123	.116	.153	1.32
.42	1.50	.057	.0865	.104	1.205	.60	2.70	.1175	.114	.149	1.305
.45	1.35	.069	.090	.114	1.265	.58	2.55	.1105	.111	.145	1.30
.48	1.90	.0785	.0965	.122	1.262	.56	2.15	.103	.1015	.140	1.30
.51	2.10	.085	.100	.127	1.27	.53	1.95	.092	.0975	.132	1.752
.53	2.20	.092	.1025	.132	1.285	.51	1.80	.085	.095	.127	1.330
.56	2.50	.103	.110	.140	1.272	.48	1.60	.0785	.080	.122	1.37
.58	2.75	.1103	.115	.1445	1.255	.45	1.43	.069	.085	.114	1.31
.60	2.90	.1175	.118	.149	1.26	.42	1.30	.057	.080	.104	1.31
.62	3.10	.123	.122	.153	1.255	.40	1.00	.0525	.069	.100	1.15
.65	3.40	.131	.1275	.158	1.24	.36	.80	.043	.0625	.0903	1.44
.69	3.60	.150	.131	.168	1.381	.35	.60	.0375	.0525	.0845	1.31
.72	3.85	.160	.135	.1745	1.292	.30	.55	.033	.050	.079	1.78
.74	4.00	.168	.138	.1785	1.292	.25	.40	.025	.0425	.069	1.22
.79	4.40	.187	.13445	.188	1.30	.20	.20	.0164	.025	.0557	2.225
.82	4.30	.197	.1475	.194	1.315	.10	0	0	0	0	
.84	4.70	.205	.149	.197	1.32						
.89	5.10	.258	.185	.2215	1.43						
.90	6.30	.262	.1725	.223	1.29						
.92	6.45	.270	.175	.227	1.297						
.90	6.30	.262	.1725	.223	1.29						
.89	6.10	.258	.170	.2215	1.305						
.86	4.65	.213	.1585	.201	1.265						
									Ave. Coef.	=	1.753
Short circuit voltage rose from											
37.5 to 38.5 and amperes from .97											
to .99 during this run.											

VII

Ar- ges	Van- turi Head in Inches of Head	Pitot Dynam- ic Head in Inches	Flow In Cubic Feet Per Second	Coef- fici- ent	Ar- ges	Van- turi Head in Inches	Pitot Dynam- ic Head in Inches	Flow In Cubic Feet Per Second	Coef- fici- ent	Ar- ges	Van- turi Head in Inches	Pitot Dynam- ic Head in Inches	Flow In Cubic Feet Per Second	Coef- fici- ent	Ar- ges
87.	32.4	081.	841.	1.81.	38.	08.4	181.	181.	1.81.	38.	08.4	181.	181.	1.81.	38.
88.	00.2	091.	41.	1.81.	39.	00.2	191.	191.	1.81.	39.	00.2	191.	191.	1.81.	39.
88.	31.2	113.	221.	1.81.	40.	31.2	221.	221.	1.81.	40.	31.2	221.	221.	1.81.	40.
88.	35.0	133.	273.	1.81.	41.	35.0	273.	273.	1.81.	41.	35.0	273.	273.	1.81.	41.
88.	00.7	153.	288.	1.81.	42.	00.7	288.	288.	1.81.	42.	00.7	288.	288.	1.81.	42.
88.	02.7	173.	303.	1.81.	43.	02.7	303.	303.	1.81.	43.	02.7	303.	303.	1.81.	43.
88.	02.0	193.	318.	1.81.	44.	02.0	318.	318.	1.81.	44.	02.0	318.	318.	1.81.	44.
88.	01.0	213.	333.	1.81.	45.	01.0	333.	333.	1.81.	45.	01.0	333.	333.	1.81.	45.
88.	38.2	233.	348.	1.81.	46.	38.2	348.	348.	1.81.	46.	38.2	348.	348.	1.81.	46.
88.	08.4	253.	363.	1.81.	47.	08.4	363.	363.	1.81.	47.	08.4	363.	363.	1.81.	47.
87.	03.4	273.	378.	1.81.	48.	03.4	378.	378.	1.81.	48.	03.4	378.	378.	1.81.	48.
87.	03.4	293.	393.	1.81.	49.	03.4	393.	393.	1.81.	49.	03.4	393.	393.	1.81.	49.
87.	05.4	313.	408.	1.81.	50.	05.4	408.	408.	1.81.	50.	05.4	408.	408.	1.81.	50.
87.	05.4	333.	423.	1.81.	51.	05.4	423.	423.	1.81.	51.	05.4	423.	423.	1.81.	51.
87.	07.8	353.	438.	1.81.	52.	07.8	438.	438.	1.81.	52.	07.8	438.	438.	1.81.	52.
88.	2.2	373.	453.	1.81.	53.	2.2	453.	453.	1.81.	53.	2.2	453.	453.	1.81.	53.
88.	00.4	393.	468.	1.81.	54.	00.4	468.	468.	1.81.	54.	00.4	468.	468.	1.81.	54.
87.	03.4	413.	483.	1.81.	55.	03.4	483.	483.	1.81.	55.	03.4	483.	483.	1.81.	55.
87.	03.4	433.	498.	1.81.	56.	03.4	498.	498.	1.81.	56.	03.4	498.	498.	1.81.	56.
87.	05.4	453.	513.	1.81.	57.	05.4	513.	513.	1.81.	57.	05.4	513.	513.	1.81.	57.
87.	05.4	473.	528.	1.81.	58.	05.4	528.	528.	1.81.	58.	05.4	528.	528.	1.81.	58.
87.	08.1	493.	543.	1.81.	59.	08.1	543.	543.	1.81.	59.	08.1	543.	543.	1.81.	59.
87.	08.1	513.	558.	1.81.	60.	08.1	558.	558.	1.81.	60.	08.1	558.	558.	1.81.	60.
87.	08.1	533.	573.	1.81.	61.	08.1	573.	573.	1.81.	61.	08.1	573.	573.	1.81.	61.
87.	08.1	553.	588.	1.81.	62.	08.1	588.	588.	1.81.	62.	08.1	588.	588.	1.81.	62.
87.	08.1	573.	603.	1.81.	63.	08.1	603.	603.	1.81.	63.	08.1	603.	603.	1.81.	63.
87.	08.1	593.	618.	1.81.	64.	08.1	618.	618.	1.81.	64.	08.1	618.	618.	1.81.	64.
87.	08.1	613.	633.	1.81.	65.	08.1	633.	633.	1.81.	65.	08.1	633.	633.	1.81.	65.
87.	08.1	633.	648.	1.81.	66.	08.1	648.	648.	1.81.	66.	08.1	648.	648.	1.81.	66.
87.	08.1	653.	663.	1.81.	67.	08.1	663.	663.	1.81.	67.	08.1	663.	663.	1.81.	67.
87.	08.1	673.	678.	1.81.	68.	08.1	678.	678.	1.81.	68.	08.1	678.	678.	1.81.	68.
87.	08.1	693.	693.	1.81.	69.	08.1	693.	693.	1.81.	69.	08.1	693.	693.	1.81.	69.
87.	08.1	713.	708.	1.81.	70.	08.1	708.	708.	1.81.	70.	08.1	708.	708.	1.81.	70.
87.	08.1	733.	723.	1.81.	71.	08.1	723.	723.	1.81.	71.	08.1	723.	723.	1.81.	71.
87.	08.1	753.	738.	1.81.	72.	08.1	738.	738.	1.81.	72.	08.1	738.	738.	1.81.	72.
87.	08.1	773.	753.	1.81.	73.	08.1	753.	753.	1.81.	73.	08.1	753.	753.	1.81.	73.
87.	08.1	793.	768.	1.81.	74.	08.1	768.	768.	1.81.	74.	08.1	768.	768.	1.81.	74.
87.	08.1	813.	783.	1.81.	75.	08.1	783.	783.	1.81.	75.	08.1	783.	783.	1.81.	75.
87.	08.1	833.	798.	1.81.	76.	08.1	798.	798.	1.81.	76.	08.1	798.	798.	1.81.	76.
87.	08.1	853.	813.	1.81.	77.	08.1	813.	813.	1.81.	77.	08.1	813.	813.	1.81.	77.
87.	08.1	873.	828.	1.81.	78.	08.1	828.	828.	1.81.	78.	08.1	828.	828.	1.81.	78.
87.	08.1	893.	843.	1.81.	79.	08.1	843.	843.	1.81.	79.	08.1	843.	843.	1.81.	79.
87.	08.1	913.	858.	1.81.	80.	08.1	858.	858.	1.81.	80.	08.1	858.	858.	1.81.	80.

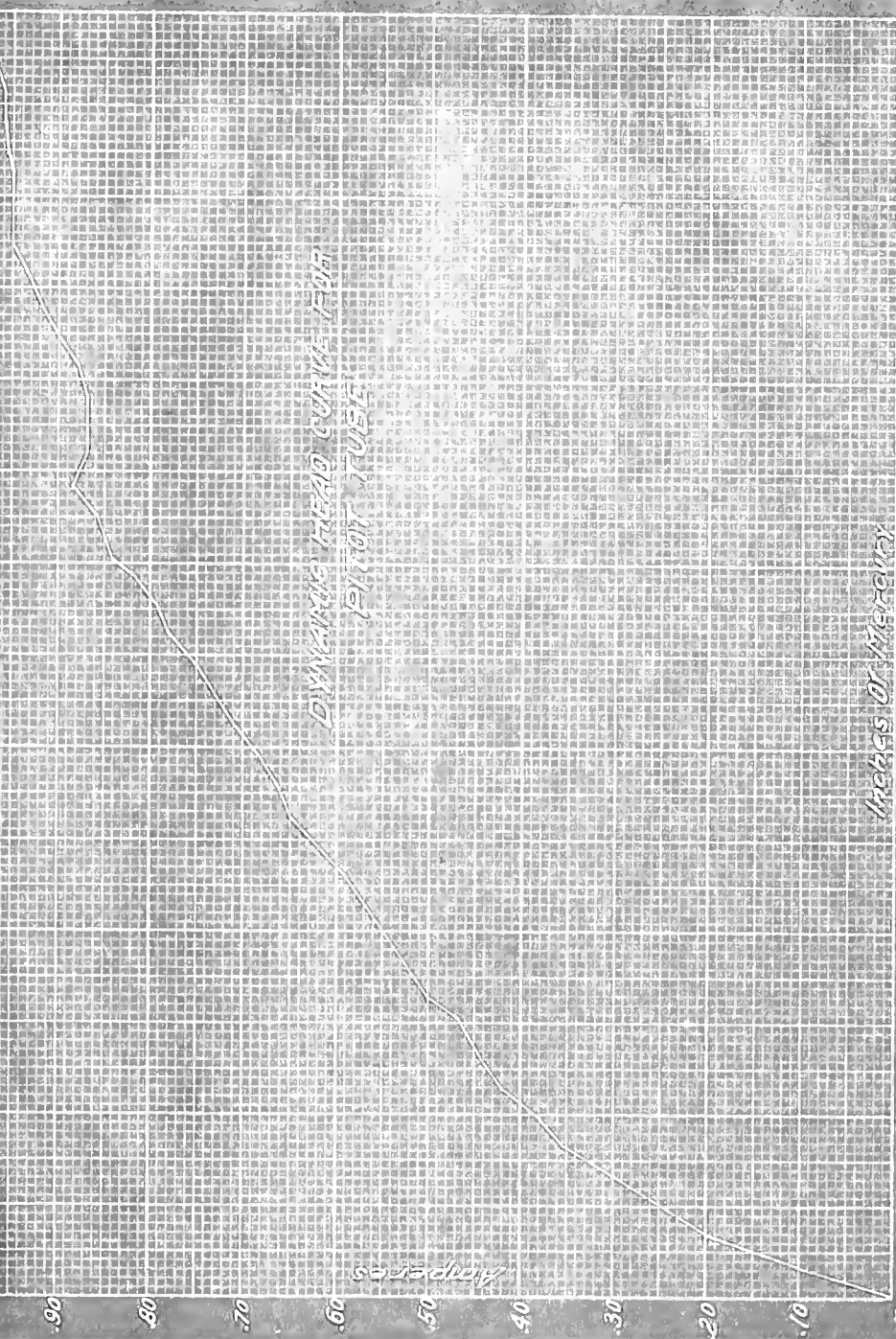
Amperes	Venturi Head in Inches of Mercury	Pitot Dynamic Head in Inches of Mercury	Flow In Cubic Feet Per Second		Coefficient	Amperes	Venturi Head in Inches of Mercury	Pitot Dynamic Head in Inches of Mercury	Flow In Cubic Feet Per Second		Coefficient
			From Venturi	From Pitot					From Venturi	From Pitot	
0	0	0	0	0		.45	1.13	.069	.0835	.114	1.365
10	.21	.0002	.025	.0395	1.58	.39	1.00	.051	.069	.0993	1.425
20	.35	.0164	.0575	.0557	1.482	.35	.90	.041	.0635	.108	1.32
30	.50	.037	.0535	.069	1.315	.30	.70	.033	.0575	.079	1.37
40	.60	.033	.0625	.079	1.265	.25	.40	.025	.0425	.019	1.62
50	1.00	.041	.069	.088	1.272	.20	.30	.0164	.034	.0557	1.635
60	1.20	.051	.0775	.0983	1.270	.10	.15	.0082	.023	.0395	1.80
70	1.35	.069	.090	.114	1.270	.0	0	0	0	0	
80	1.6	.082	.100	.1247	1.247						
90	2.50	.090	.110	.131	1.192						
100	3.00	.115	.1195	.1478	1.235		Ave.	coef.	=	1.36	
110	3.60	.1405	.131	.1665	1.27						
120	4.20	.164	.141	.1782	1.25						
130	4.95	.180	.148	.185	1.252						
140	5.00	.199	.154	.1942	1.26						
150	5.17	.217	.153	.2015	1.29						
160	5.25	.234	.1573	.219	1.30						
170	7.00	.252	.1625	.2215	1.215						
180	7.50	.270	.169	.2265	1.20						
190	8.30	.258	.177	.2215	1.252						
200	9.10	.254	.179	.219	1.285						
210	5.85	.213	.1665	.2015	1.21						
220	4.80	.199	.151	.1942	1.285						
230	4.20	.180	.141	.185	1.31						
240	4.00	.164	.1375	.1763	1.28						
250	3.35	.1375	.103	.162	1.285						
260	2.70	.115	.114	.1478	1.30						
270	2.30	.090	.105	.131	1.25						
280	1.80	.083	.095	.1247	1.315						



40
35
30
25
20
15
10
5
4
3
2
1

Plate 2

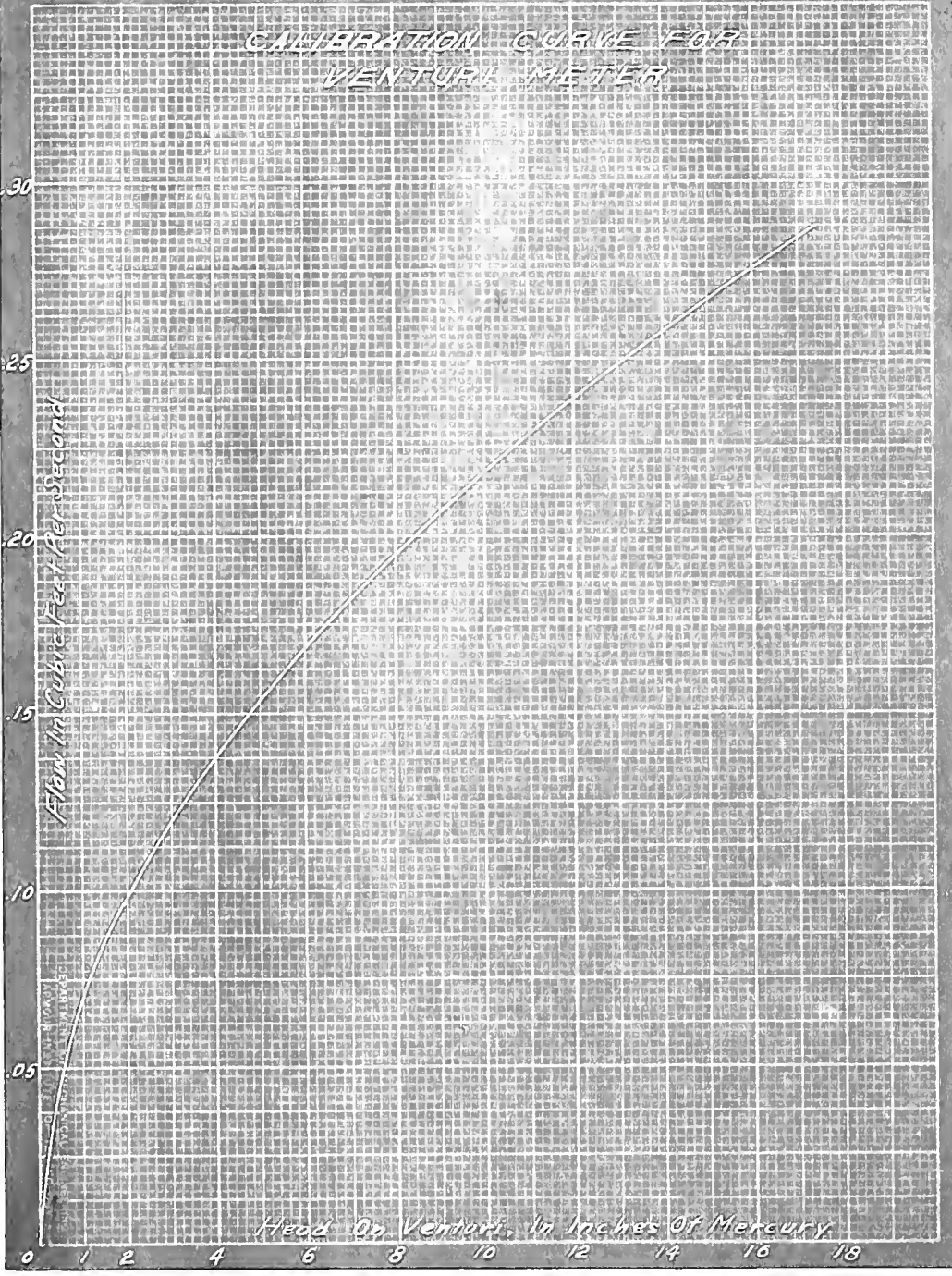
100
90
80
70
60
50
40
30
20
10
0



100
90
80
70
60
50
40
30
20
10
0

Plate 3

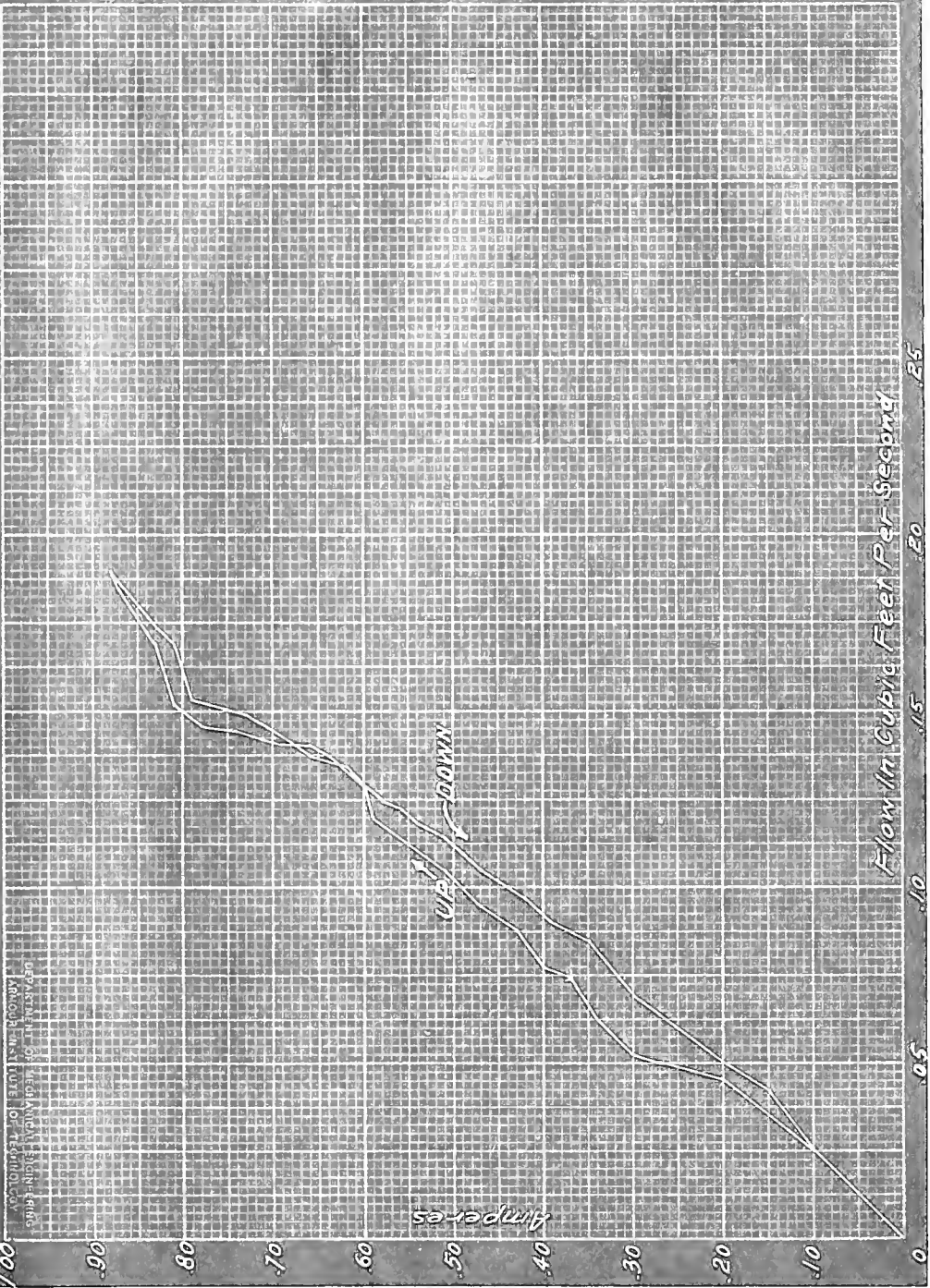
CONVERSION CHART FOR VENTURI METER



Flow In Cubic Feet Per Second

Head On Venturi, In Inches Of Mercury

Plate 4



Flow in Cubic Feet Per Second

Amperes

UP
DOWN

1.00
0.90
0.80
0.70
0.60
0.50
0.40
0.30
0.20
0.10
0

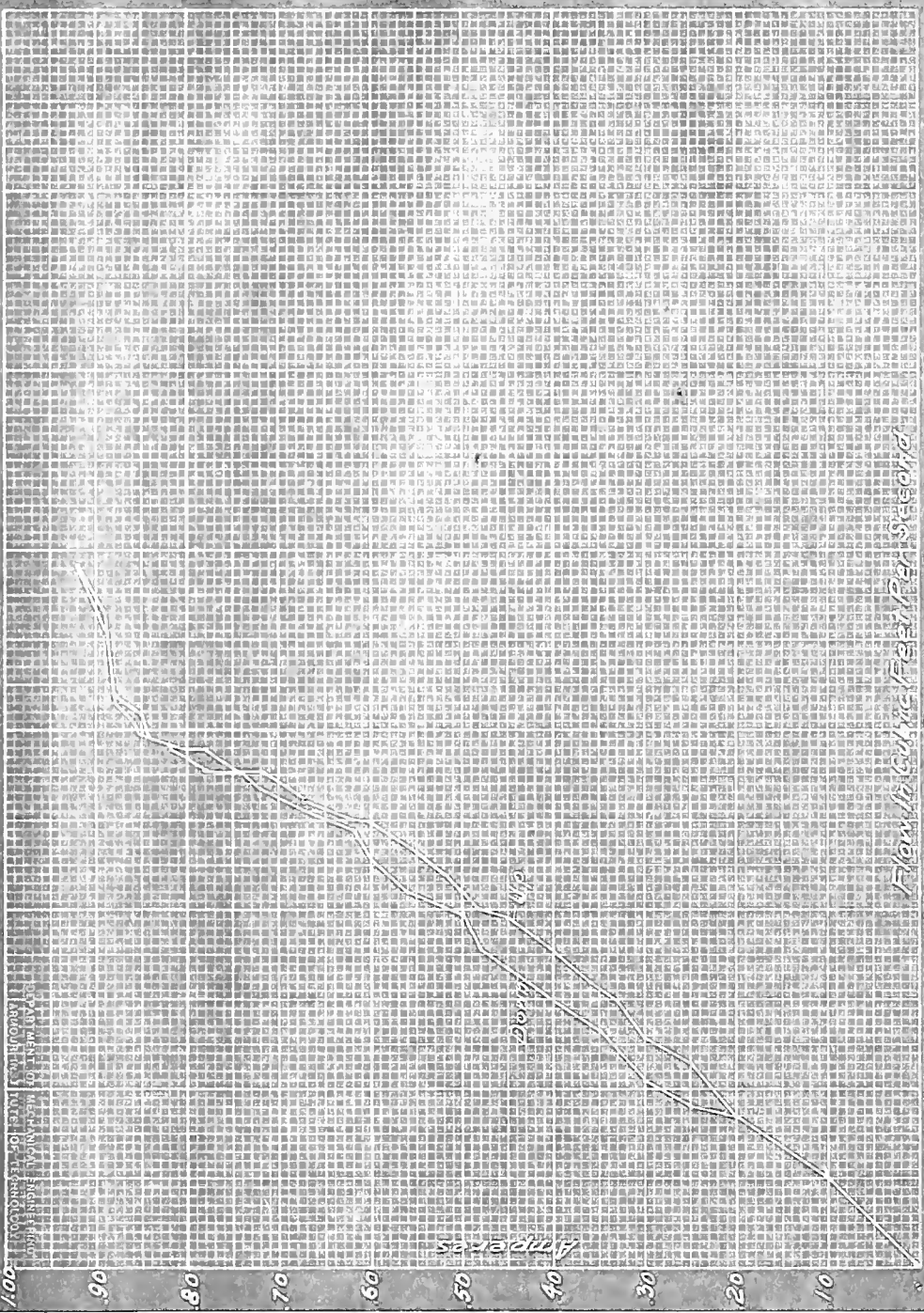
0
0.05
0.10
0.15
0.20
0.25

Plate 5.



Forth Co. 25 20 15 10 05

Plate 6



100
90
80
70
60
50
40
30
20
10
0

10
0.5
1.0
1.5
2.0
2.5
3.0
3.5
4.0
4.5
5.0
5.5
6.0
6.5
7.0
7.5
8.0
8.5
9.0
9.5
10

0

propeller

Flow to Starboard

Plate 7.



Flow in Cubic Feet per Second

1.00 0.90 0.80 0.70 0.60 0.50 0.40 0.30 0.20 0.10 0

0.05

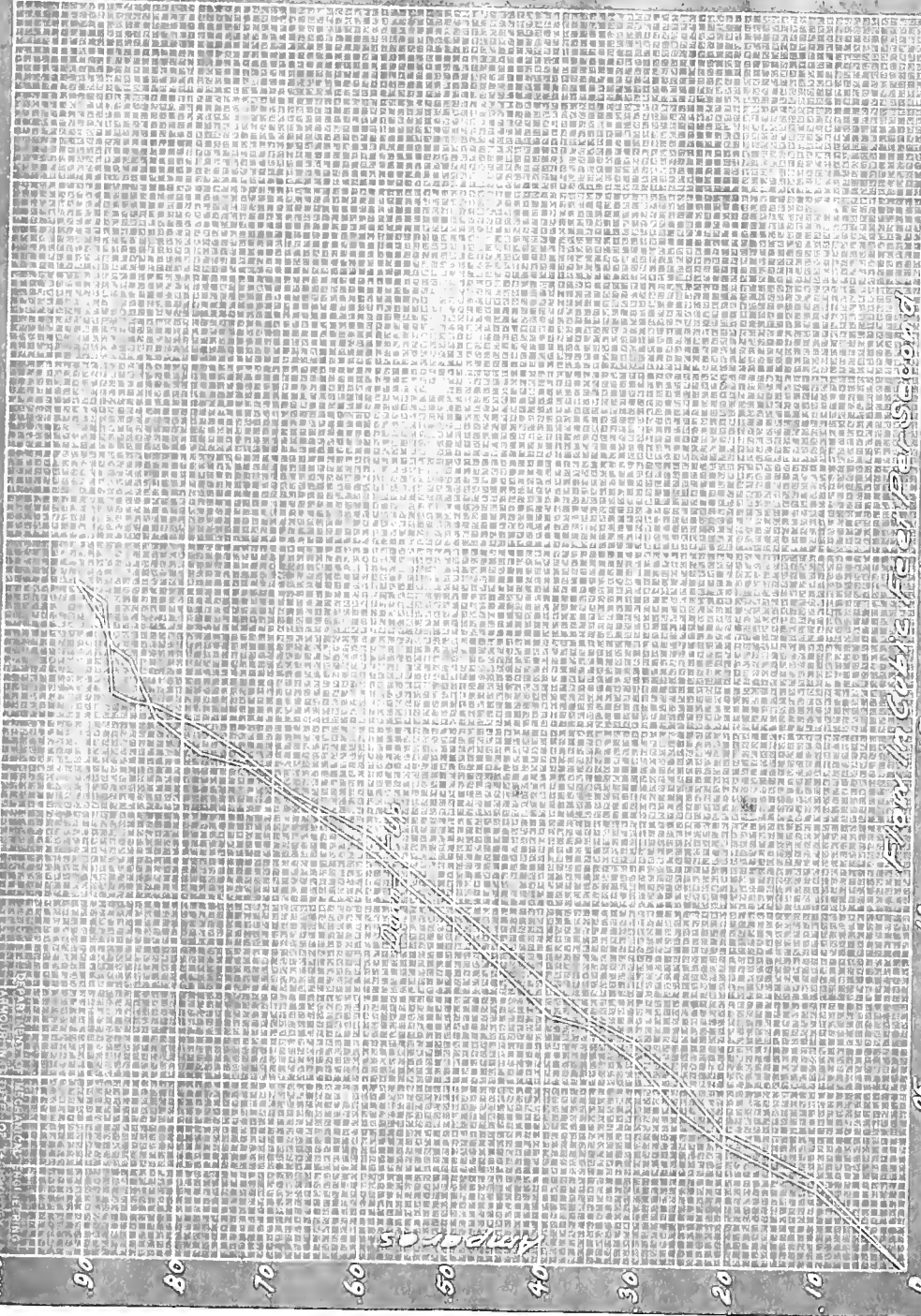
0.10

0.15

0.20

0.25

Plate 8



1500 ft. above sea level

100 90 80 70 60 50 40 30 20 10 0

05

10

15

20

25

Plate 9.

2.0
1.9
1.8
1.7
1.6
1.5
1.4
1.3
1.2
1.1
1.0
0

11.0 0 1.05 1.10 1.15 1.20 1.25

Flow in cubic feet per second



Plat e 10.

20
19
18
17
16
15
14
13
12
11
10

20
15
10
05

North Capitol Street, S. E. corner

Ohio

Ave 111

Copyright 1914

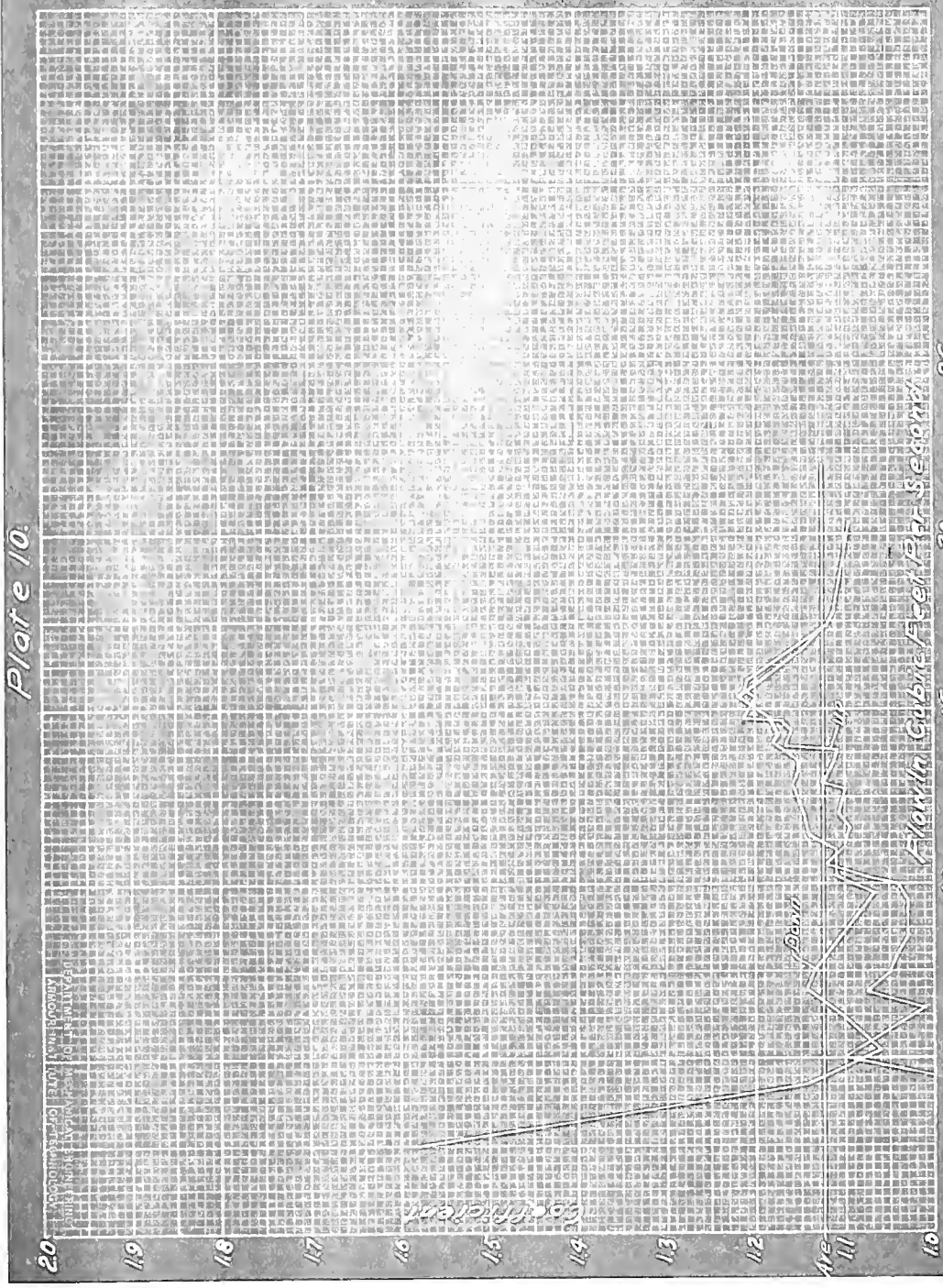


Plate II.

2.0
1.9
1.8
1.7
1.6
1.5
1.4
1.3
1.2
1.1

Coefficient

1.0
0.5

Flow in Cubic Feet Per Second

2.5
2.0
1.5
1.0
0.5

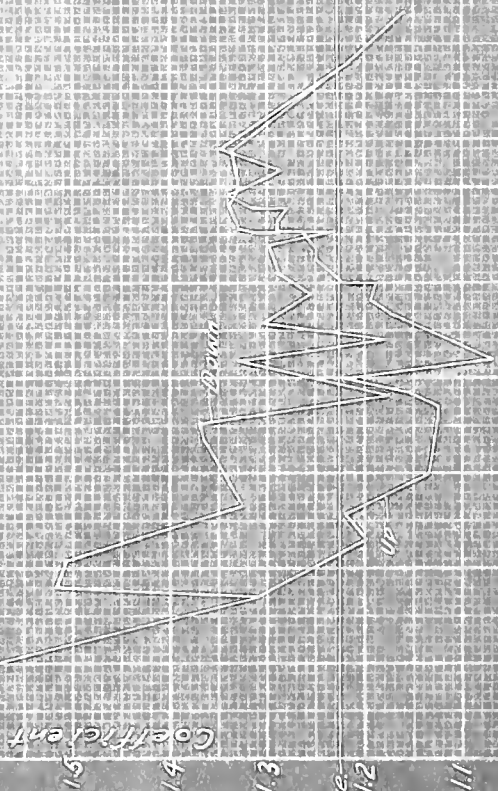


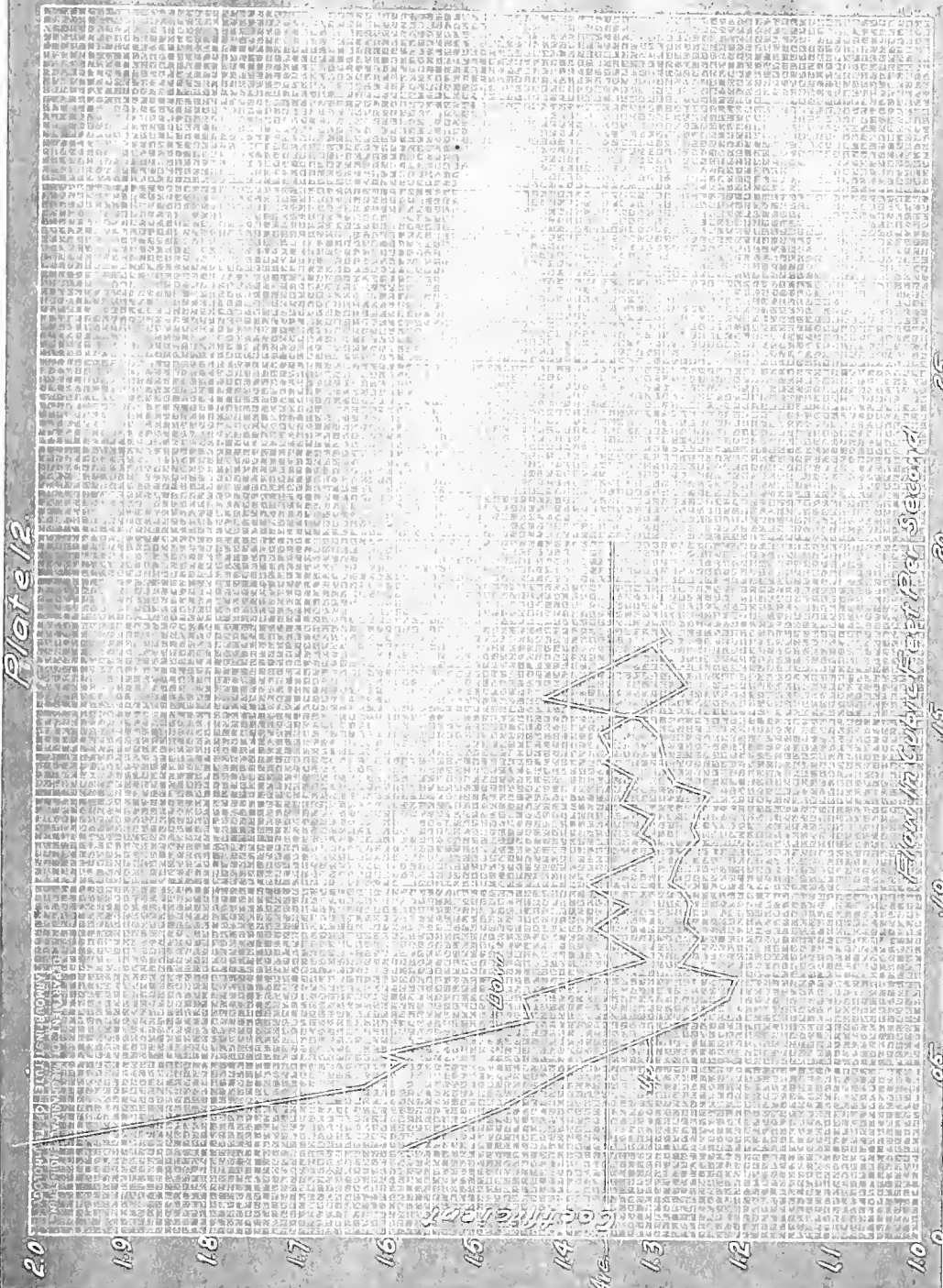
Plate 12

Flow in Multiple Jet Section

100 110 120 130 140 150 160 170 180 190 200

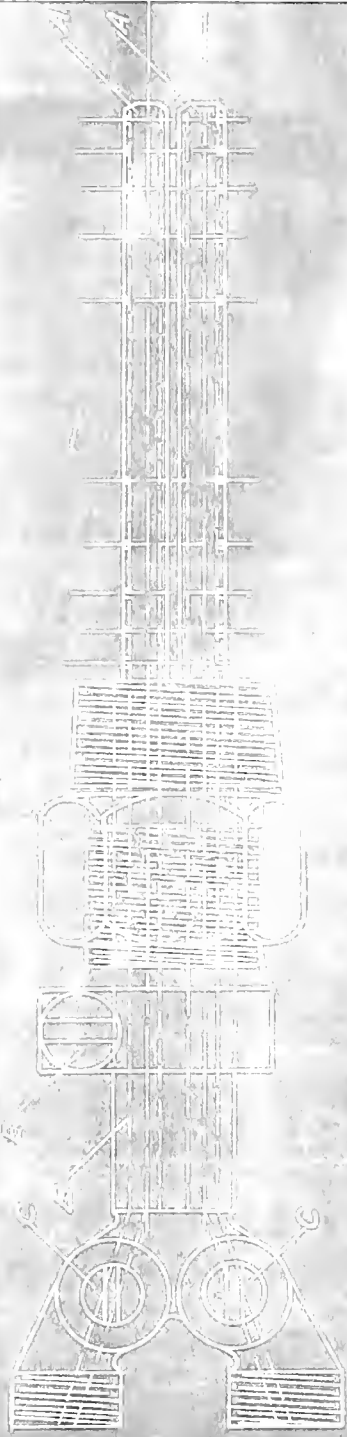
2.0 1.9 1.8 1.7 1.6 1.5 1.4 1.3 1.2 1.1 1.0

Coffinrock



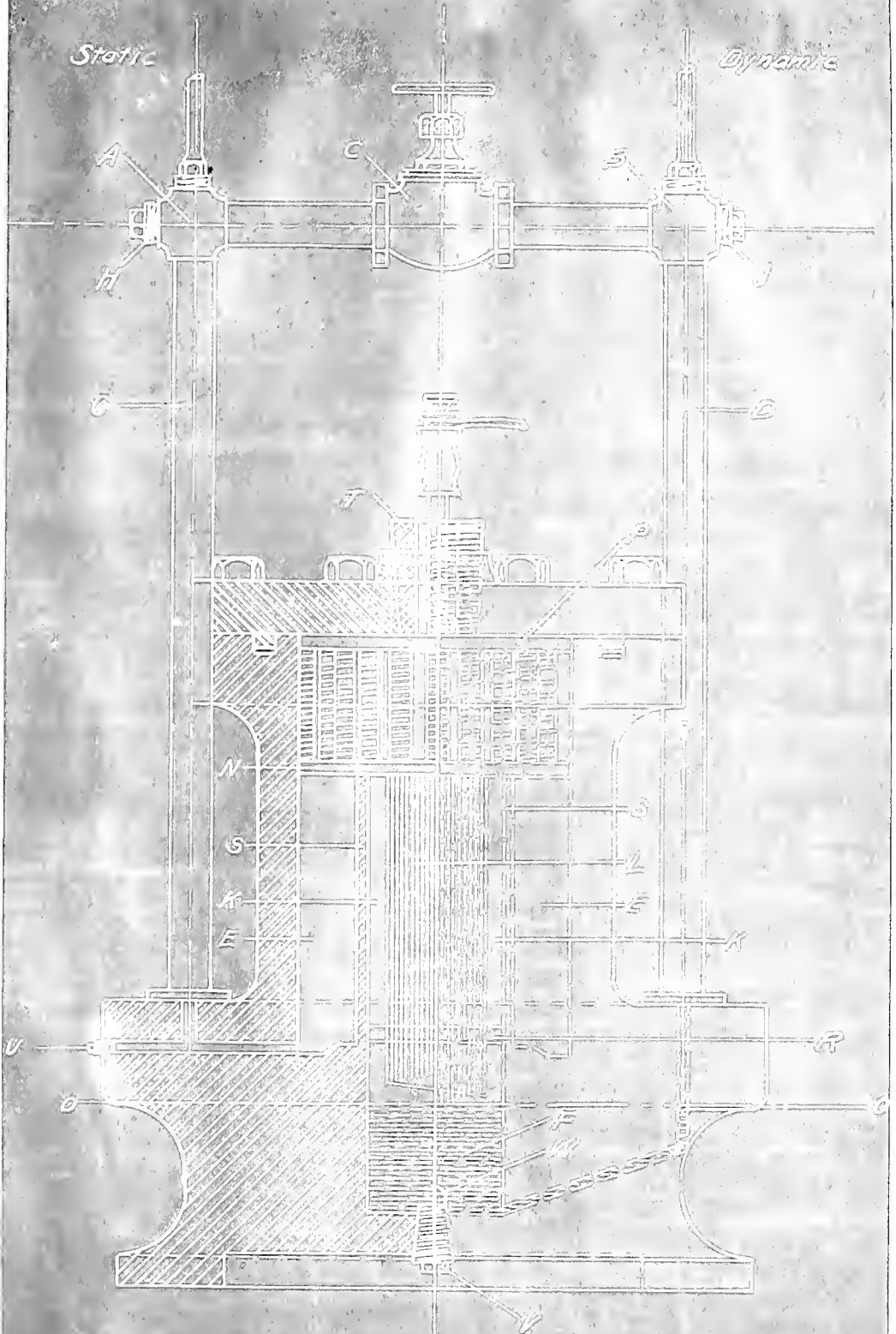
Plat 17

17



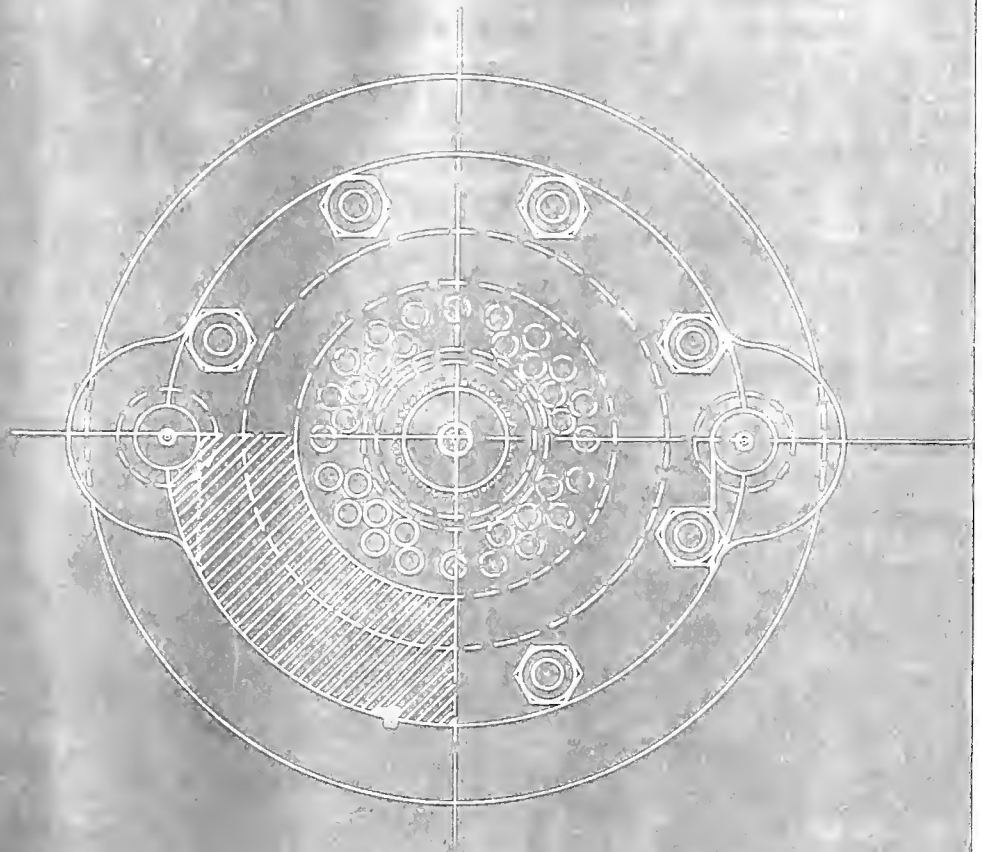
THE MULTITROPENING AIR-RAISING PILOT TUBE.

Scale—Full Size



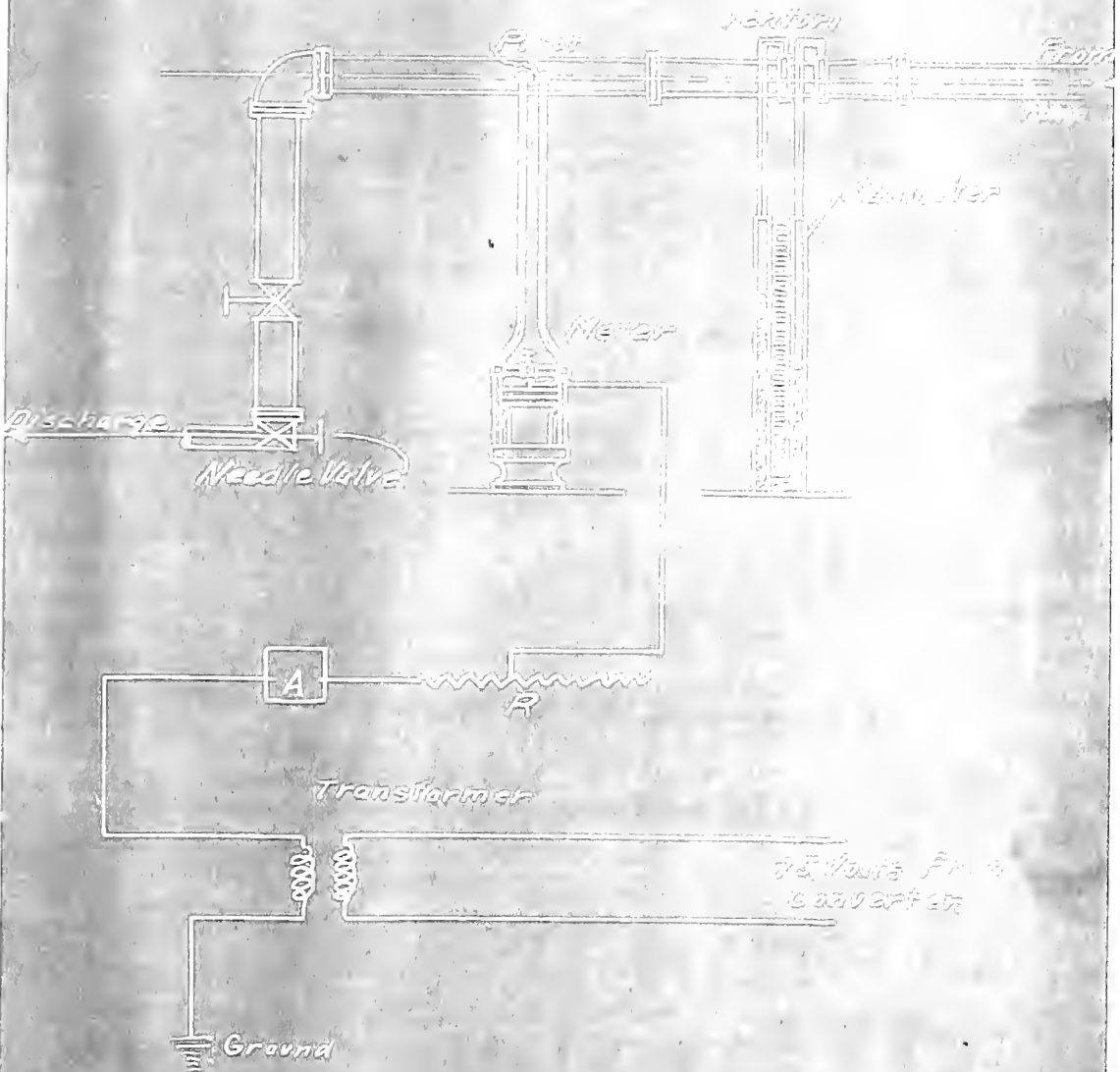
SECTION THROUGH METER
Scale - Half Size

Plate 16.



*SECTION THROUGH METER.
Scale - Half Size*

Plate 17



Scheme Of Connections For Test





